

## Screw Theory

Displacement &  
Twist

Force & Wrench

Screws in Plücker  
Coordinates

Group Theory

A short treatise on robots' kinematic geometry and kinetics.

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June 09, 2022

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# Lecture IV Outline

## Screw Theory

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## Lecture IV Outline

### Screws Theory and Rigid Body Transformations.

Screws (properly revisited): Chasles' and Poincaré's theorem; Displacement and Force screws; Plücker coordinates.

Wrench; Instantaneous screw axis; Couple; Adjoint maps; Velocity transformations – in Body and Spatial Homogeneous Coordinates.

Group theory: The Lie algebra, motions in  $\mathfrak{se}(3)$ , and the Lie Group.

Manipulator kinematics: Brockett's exponential map formula. Paden-Kahan subproblems. Denavit-Hartenberg Conventions. *short treatise on robots' kinematics*

# Rigid Body Motions as Screws

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## Rigid Body Motion as a Screw Motion

The motion of a **rigid body** is precisely the same as if it were attached to the **nut of a literal mechanical screw**. Associated with the screw is its pitch.

## Definition (Screw)

That straight line with which a **definite linear magnitude** termed the pitch is associated is called the **screw**.

# Screw as a Geometric Quantity

## Screw Theory

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## Pitch of a Screw

The **rectilinear distance** through which (a literal nut) **nut is translated parallel to the axis of a screw**, while the nut is rotated through the **angular unit of circular measure** is termed the **pitch**.

## Plücker Coordinates

Let  $\mathbf{a}$  be a point on line  $\ell_0$ . Let  $\mathbf{a}$ 's direction cosine vector (to be introduced shortly) be  $\mathbf{b}$ . Then, its binormal (moment) vector is  $\mathbf{c} = \mathbf{a} \times \mathbf{b}$ . We say the pair  $(\mathbf{b}, \mathbf{c})$  is the **Plücker Coordinates** of point  $\mathbf{a}$  on axis  $\ell_0$ .

# Screw in Plücker Coordinates

## Screw Theory

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Screws in Plücker

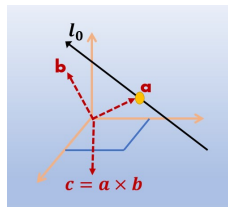
Coordinates

Group Theory

### Definition (Screw Coordinates)

Six-vector,  $s$ , related to the Plücker coordinates, parameterize a screw i.e.

$$s = (s_1, s_2, s_3, s_4, s_5, s_6).$$



# Screws and Plücker Coordinates

## Screw Theory

### Displacement & Twist

### Force & Wrench

### Screws in Plücker Coordinates

### Group Theory

## Screw axis and Plücker Coordinates

$$b_1 = s_1, \quad b_2 = s_2, \quad b_3 = s_3; \quad (1)$$

$$c_1 = s_4 - p \cdot s_1, \quad c_2 = s_5 - p \cdot s_2, \quad c_3 = s_6 - p \cdot s_3. \quad (2)$$

## Pitch in Plücker Coordinates

$$p = \frac{s_1 s_4 + s_2 s_5 + s_3 s_6}{s_1^2 + s_2^2 + s_3^2}, \quad (3)$$

$$|s| = \sqrt{s_1^2 + s_2^2 + s_3^2} \quad \text{if } p \neq \infty, \quad (4)$$

$$|s| = \sqrt{s_4^2 + s_5^2 + s_6^2} \quad \text{if } p = \infty \quad (5)$$

# Pitch and Magnitude of the screw

## Screw Theory

### Displacement & Twist

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## Plücker Coordinates' Direction Cosines

Suppose that  $h = \sqrt{b_1^2 + b_2^2 + b_3^2}$ . Then  $(\mathbf{b}/h, \mathbf{c}/h)$  are respectively the direction cosines of the line,  $l_0$  and its moment.

## Homogeneous Coordinates!

**Plücker Coordinates** give six unit parameters of a point on a line. Plücker Coordinates are in **homogeneous coordinates**!



# Twist About a Screw (Axis)

## Screw Theory

### Displacement & Twist

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## Twist

A body's **twist** about a **screw** is a **uniform (infinitesimal) rotation** about the screw **followed by a uniform (infinitesimal) translation** about an **axis parallel to the screw**, through a **distance that is the product of the pitch and the circular measure of rotation**.

## Twist

A **twist** requires six **algebraic quantities** for its **complete specification**: **five** ( $\{t_i\}_{i=1}^5$ ) specify the screw, the **sixth (or its amplitude)** specifies the **screw's rotaty angle**,  $t_6$ .

# Twist in Plücker Coordinates

## Screw Theory

### Displacement & Twist

#### Force & Wrench

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#### Group Theory

## Definition (Twist Coordinates)

A six-vector,  $\mathbf{t}$ , related to the Plücker coordinates parameterize a twist vector i.e.  $\mathbf{t} = [(t_1, t_2, t_3), (t_4, t_5, t_6)]$  or  $\mathbf{t} = (\boldsymbol{\omega}, \mathbf{v})$ , where  $\boldsymbol{\omega} = (t_1, t_2, t_3)$  and  $\mathbf{v} = (t_4, t_5, t_6)$ .

## Plücker Coordinates of a Twist

$$b_1 = t_1, \quad b_2 = t_2, \quad b_3 = t_3 \quad (6)$$

$$c_1 = t_4 - p \cdot s_1, \quad c_2 = t_5 - p \cdot s_2, \quad c_3 = t_6 - p \cdot s_3. \quad (7)$$

# Twists in Plücker Coordinates

## Screw Theory

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## Pitch of the Twist

$$p_t = \frac{t_1 t_4 + t_2 t_5 + t_3 t_6}{t_1^2 + t_2^2 + t_3^2} = \frac{\boldsymbol{\omega} \cdot \boldsymbol{v}}{\boldsymbol{\omega} \cdot \boldsymbol{\omega}}$$

## Pitch of the Twist

Expressed as a ratio of the **magnitude of the velocity of a point on the twist axis** to the **magnitude of the angular velocity** about the twist axis.

## Translation Distance

$d_t = t_6 \times p_t$ . The sign expresses the rotation's direction.

# Twists and Fixed Movements

## Screw Theory

### Displacement & Twist

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### Pure Rotation

Let pitch be **zero**. That which results is but **pure rotation**.

### Pure Translation

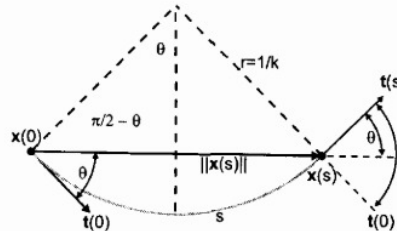
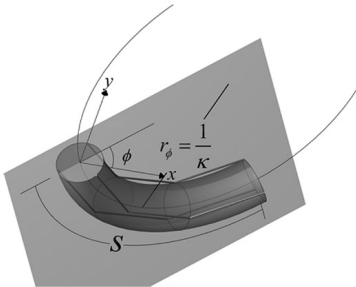
Let pitch be **infinite**. That which results **cannot be a finite twist**, **except the amplitude be zero**, whereupon the **twist becomes a pure translation parallel to the screw**.

# Curvilinear Displacement: Serret-Frenet Frame

## Screw Theory

### Displacement & Twist

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Elephant Trunk Multi-sectional Continuum Model (left), and its Representation in the Serret-Frenet Frame.

# Plücker Coordinates Example

## Screw Theory

### Displacement & Twist

### Force & Wrench

### Screws in Plücker Coordinates

### Group Theory

## Chasles' Theorem Applied to The Serret-Frenet Frame

Consider a spatial curve  $\mathcal{S}$  on the elephant continuum trunk shown earlier. Suppose  $\mathcal{S}$  is parameterized by its arc length  $s \in [0, 1]$ . For a point  $\mathbf{x} = [x, y, z]^T$  on  $\mathcal{S}$ , the unit tangent vector at  $s$  is  $\mathbf{t}(s) = d\mathbf{x}/ds$ .

## Differential Kinematics and The Serret-Frenet Frame

Denote by  $\mathbf{n}$  the principal normal to  $\mathcal{S}$  at  $\mathbf{n}$ ; then we must have  $\mathbf{b} = \mathbf{t} \times \mathbf{n}$  as the binormal. We say  $(\mathbf{b}, \mathbf{n})$  is the Plücker coordinate of the tangent  $\mathbf{t}$ .

# Force

## Screw Theory

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**Force & Wrench**

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## Force

Net **force** exerted on a body,  $\mathbf{F} = (f_x, f_y, f_z)$ .

## Couple of Force

Suppose that  $\mathbf{F}$  acts along a corkscrew axis. The resulting motion when  $\mathbf{F}$  makes an infinitesimal rotation about its screw axis is called its **couple**,  $\mathfrak{C} = (c_x, c_y, c_z)$ .

# Complete Wrench on a Screw

## Screw Theory

Displacement &  
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## Wrench

A wrench requires six algebraic quantities for its complete specification: five  $(\{w_i\}_{i=1}^5)$  specify the screw, the sixth (or its intensity),  $w_6$ , specifies the force's magnitude.

## Couple's Moment

The moment of the couple is the product of the intensity of the wrench and the screw's pitch i.e.

$$\alpha(\mathcal{C}) = w_6 \times p_w.$$



# Wrench on a Screw

## Screw Theory

Displacement &  
Twist

### Force & Wrench

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## Wrench

Simple Definition: A **force** and a **couple** both acting in a plane perpendicular to the force.

## Definition (Complete Definition)

The **resultant canonical system of forces** acting on a rigid body, **reduced to a resultant force on a point**, and acting along the **resultant couple** that is **perpendicular to the plane** in which the force acts is called **the wrench**.

# Wrench in Plücker Coordinates

## Screw Theory

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## Definition (Wrench Coordinates)

A six-vector,  $\mathbf{w}$ , related to the Plücker coordinates parameterize a wrench vector i.e.

$\mathbf{w} = [(w_1, w_2, w_3), (w_4, w_5, w_6)]$  or  $\mathbf{w} = (\mathbf{f}, \mathbf{m})$ , where  $\mathbf{f} = (w_1, w_2, w_3)$  and  $\mathbf{m} = (w_4, w_5, w_6)$ .

## Plücker Coordinates of a Wrench

$$b_1 = w_1, \quad b_2 = w_2, \quad b_3 = w_3 \quad (8)$$

$$c_1 = w_4 - p \cdot s_1, \quad c_2 = w_5 - p \cdot s_2, \quad c_3 = w_6 - p \cdot w_3. \quad (9)$$

# Wrench in Plücker Coordinates

## Screw Theory

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## Pitch of the Wrench

$$p_t = \frac{w_1 w_4 + w_2 w_5 + w_3 w_6}{w_1^2 + w_2^2 + w_3^2} = \frac{\mathbf{f} \cdot \mathbf{m}}{\mathbf{f} \cdot \mathbf{f}}$$

## Pitch of the Wrench

Expressed as a ratio of the **moment applied about a point on the axis** to the **magnitude of the force applied** along the wrench axis.

## Wrench's Magnitude

$$\begin{aligned} \|\mathbf{f}\| &= \sqrt{w_1^2 + w_2^2 + w_3^2} \text{ if } p_w = 0 \text{ else} \\ \|\mathbf{m}\| &= \sqrt{w_4^2 + w_5^2 + w_6^2} \text{ if } p_w = \infty. \end{aligned}$$

# Wrenches and Fixed Movements

## Screw Theory

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## Pure Force

Let pitch be **zero**. That which results is **pure force** along its screw axis.

## Pure Couple

Let pitch be **infinite**. That which results **cannot be a finite wrench**, **except the intensity be zero**, whereupon the wrench becomes a pure couple in a plane that is perpendicular to the screw.

# Statics and Instantaneous Kinematics

## Screw Theory

Displacement &  
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### Force & Wrench

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## Statics and kinematics

Statics	Instantaneous Kinematics
Force, $\mathbf{F}$ about $n$ .	Infinitesimal rotation, $\omega$
Couple, $\mathcal{C}$ : $[\mathbf{F}] \times [\ell]$	Infinitesimal translation, $t$
$p_w = \pm \mathcal{C} / \mathbf{F}$	Pitch of a Wrench, $w$
$ \mathbf{F} $	Intensity of Wrench

Dyname:  $(\mathbf{F}, \mathcal{C})$ . Credits: Plücker (1866), Routh (1892).

# Plücker Coordinates Kinetics Quiz

## Screw Theory

Displacement &

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## Poinsot's Theorem Quiz on a Force and its Moment

Suppose that a force  $\mathbf{F}$  acts at the point  $\mathbf{a}$  in the image of Frame 6. What are the Plücker coordinates of the **line of force**?

# A short treatise on robots' kinematic geometry and kinetics.

## └ Screw Theory

### └ Screws in Plücker Coordinates

### └ Plücker Coordinates Kinetics Quiz

#### Poinsot's Theorem Quiz on a Force and its Moment

Suppose that a force  $\mathbf{F}$  acts at the point  $\mathbf{a}$  in the image of Frame 6. What are the Plücker coordinates of the **line of force**?

## Poinsot's Theorem Quiz on a Force and its Moment

Imagine that a force  $\mathbf{F}$  is acting at the point  $\mathbf{a}$  in the image of Frame 6. Suppose that  $\boldsymbol{\tau}$  is torque acting along the normal to point  $\mathbf{a}$ . Then  $(\mathbf{f}, \boldsymbol{\tau})$  are the Plücker coordinates of the **line of force**.

## Arithmetics on Screws

Scalar and vector arithmetic operations are valid on infinitesimal screws e.g.

$$c_1 \mathbf{s}_1 + c_2 \mathbf{s}_2 = 0 \text{ for } c_1, c_2 \neq 0 \text{ on screws } \mathbf{s}_1, \mathbf{s}_2. \quad (10)$$

# Plücker Coordinates Kinetics Quiz

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## Poinsot's Theorem Quiz on a Force and its Moment

Suppose that a force  $\mathbf{F}$  acts at the point  $\mathbf{a}$  in the image of Frame 6. What are the Plücker coordinates of the **line of force**?



# Group Theory Review

## Screw Theory

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## The Euclidean Motion

Let  $\mathbb{E}^3$  denote the ordinary Cartesian 3-space that admits the standard inner product

$$\langle x, y \rangle = \sum_i x_i y_i. \quad (11)$$

## Transformations

The set of all **length-preserving transformations** in  $\mathbb{E}^3$  shall be denoted by  $\mathbb{E}(3) \in \mathbb{R}^6$  *i.e.*, the family of **translations and rotations**<sup>a</sup>.

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<sup>a</sup>Rotations in  $\mathbb{E}^3$  are not necessarily proper.

# Group Transformation Isomorphism

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## Brockett, 1990

Euclidean transformation under group composition and Euclidean transformation under group multiplication preserve the **isomorphic property**.

## Example: Affine Euclidean Transformations

$q$  defines a Euclidean affine transformation  $q = R\mathbf{x} + \mathbf{d}$  if  $\langle R, R^T \rangle = I$  for  $(q, \mathbf{d}) \in \mathbb{R}^3$ . Now, suppose  $q = R_1\mathbf{x} + \mathbf{d}_1$  and  $p = R_2q + \mathbf{d}_2$ , then  $p = R_2R_1\mathbf{x} + \mathbf{d}_2$ .

# Group Transformation Isomorphism

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### Example: Euclidean Transformation Identity

$$\begin{pmatrix} \mathbf{R}_2 & \mathbf{d}_2 \\ 0 & 1 \end{pmatrix} \begin{pmatrix} \mathbf{R}_1 & \mathbf{d}_1 \\ 0 & 1 \end{pmatrix} = \begin{pmatrix} \mathbf{R}_2 \mathbf{R}_1 & \mathbf{R}_2 \mathbf{d}_1 + \mathbf{d}_2 \\ 0 & 1 \end{pmatrix} \quad (12)$$

### The isomorphic property (Brockett, 1990)

That matrices of the form (SE(3) matrices):  $\begin{pmatrix} \mathbf{R} & \mathbf{d} \\ 0 & 1 \end{pmatrix}$  are isomorphic.

# The General Linear Group

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## $SO(3)$ as a General Linear Group

The special orthogonal group,  $SO(3)$ , is a subgroup of the general linear group

$$SO(3) = \{\mathbf{R} \in GL(n, \mathbb{R}) : \mathbf{R}\mathbf{R}^T = \mathbf{I}, \det \mathbf{R} = 1\}. \quad (13)$$

# The Lie Group

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## The Lie Group

A group with a **topology operation** on its set of elements such that the group can be given the **structure of a differential manifold** with the property that **group multiplication and inversion** is **continuous** is called a **Lie group**.

## The Special Euclidean Matrix Group, $SE(3)$

$SE(3)$  is a **differentiable manifold**, comprised of all the **translations** and **proper rotations** that **moves a body from one point to another** in the ordinary cartesian 3-space  $\mathbf{E}^3$ .

# The Lie Group

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## The Special Euclidean Matrix Group, $SE(3)$

$$g = \begin{bmatrix} \mathbf{R} & \mathbf{d} \\ \mathbf{0}^T & 1 \end{bmatrix}; g \in SE(3). \quad (14)$$

# The Lie Group

## Screw Theory

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## The Special Euclidean Matrix Group, $SE(3)$

$$SE(3) = \{(\mathbf{R}, d) : \mathbf{R} \in SO(3), d \in \mathbb{R}^3\} := SO(3) \times \mathbb{R}^3. \quad (15)$$

I have followed **Chasles' notation**, who posited that **any rigid motion can be formed via a rotation, followed by a translation**, and that **the rotation and the translation commute** i.e.  $\mathbf{R}d = d$ .

## The Special Euclidean Matrix Group, $SE(3)$

Note: Most authors' notation follow **Euclid's theorem** i.e. any rigid motion is a **translation** followed by a **rotation about an axis** that **passes through a pre-specified (fixed) point**.

$$SE(3) = \{(d, \mathbf{R}) : d \in \mathbb{R}^3, \mathbf{R} \in SO(3)\} := \mathbb{R}^3 \times SO(3). \quad (16)$$

# The Lie Group

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## The Special Euclidean Matrix Group, $SE(3)$

Chasles' notation allows for motion representation in form of screw motions.

## Commutativity of group operations on $SE(3)$

[Brockett, 1990]: Equation (15) imply that the Lie group is a semidirect product of simple Lie subgroup of orthogonal transformations and the abelian Lie subgroup of all translations.



# The Lie Algebra

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## The Lie Algebra, $\mathfrak{se}(3)$

The Lie algebra is a vector space  $\hat{\xi}$  with the antisymmetric bilinear operation  $[\cdot, \cdot] : \hat{\xi} \times \hat{\xi} \rightarrow \hat{\xi}$  which satisfies the Jacobi identity,

$$[\hat{\xi}_1, [\hat{\xi}_2, \hat{\xi}_3]] + [\hat{\xi}_2, [\hat{\xi}_3, \hat{\xi}_1]] + [\hat{\xi}_3, [\hat{\xi}_1, \hat{\xi}_2]] = 0. \quad (17)$$

NB:  $[\cdot, \cdot]$  is alternatively the Lie bracket notation with antisymmetry operation  $[\hat{\xi}_2, \hat{\xi}_3] = -[\hat{\xi}_3, \hat{\xi}_2]$ .

# The Lie Algebra Representation

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## The Lie Algebra Representation, $\mathfrak{se}(3)$

The Lie algebra admits the following **homogeneous coordinates representation** for a point  $q \in \mathbb{R}^3$  on a link that rotates with unit velocity  $\omega$ ,

$$\hat{\xi} = \begin{pmatrix} \tilde{\omega} & v \\ 0 & 0 \end{pmatrix} \in \mathfrak{se}(3), \quad \xi = (\omega^T, v^T)^T \in \mathbb{R}^6 \quad (18)$$

where  $v = -\omega \times q$ .

# The Lie Algebra Representation

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## The Lie Algebra Representation, $\mathfrak{se}(3)$

Observe:

$$\tilde{\omega} = \begin{pmatrix} 0 & -\omega_z & \omega_y \\ \omega_z & 0 & -\omega_x \\ -\omega_y & \omega_x & 0 \end{pmatrix} \equiv -\tilde{\omega}^T \in \mathfrak{so}(3) \quad (19)$$

is the skew-symmetric form of the velocity of the tip point,  $\omega \in \mathbb{R}^3$ .

# The Lie Algebra Diffeomorphisms

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## Lie Representation Snippet

Observe:

$$\tilde{(\cdot)}_{SO(3)} : \mathbb{R}^3 \rightarrow \mathfrak{so}(3) \quad (20)$$

$$\tilde{(\cdot)}_{SE(3)} : \mathbb{R}^6 \rightarrow \mathfrak{se}(3) \quad (21)$$

$\tilde{\omega}(S) \in \mathfrak{se}(3)$ : e.g. **Twist parameterization** of a curve, deformation, **screw**.

$\omega(S) \in \mathbb{R}^6$ : e.g. **Motion vector** e.g. linear + angular velocities, axial, shear, bending, and torsion motion.

# The exponential map belongs to the Lie Group

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The exponential map,  $\exp(\mathfrak{se}(3))$ , is an element of  $SE(3)$

Given  $g : \begin{pmatrix} \mathbf{R}(\theta) & \mathbf{d} \\ 0 & 1 \end{pmatrix} \in SE(3)$  there exists a  $\tilde{\xi} = (\tilde{\omega}, v) \in \mathfrak{se}(3)$ , such that  $\exp(\tilde{\xi}\theta) \in SE(3)^a$ .

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<sup>a</sup>Proof in Murray and Sastry, Prop 2.8.

The exponential map,  $\exp(\mathfrak{se}(3))$ , is surjective onto  $SE(3)$

Given  $g : \begin{pmatrix} \mathbf{R}(\theta) & \mathbf{d} \\ 0 & 1 \end{pmatrix} \in SE(3)$  there exists a  $\begin{pmatrix} \tilde{\omega} & \mathbf{d} \\ 0 & 0 \end{pmatrix}; \tilde{\omega} = -\tilde{\omega}^T$ , such that  $\exp(\tilde{\omega}) = g^a$ .

---

<sup>a</sup>Proof in Murray and Sastry, Prop 2.9.

# Chasles and Affine Transformations

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## Chasles Theorem and Affine Transformations

$$\begin{pmatrix} \mathbf{R} & \mathbf{d} \\ 0 & 1 \end{pmatrix} = \begin{pmatrix} \mathbf{I} & \mathbf{c} \\ 0 & 1 \end{pmatrix} \begin{pmatrix} \mathbf{R} & \mathbf{d} \\ 0 & 1 \end{pmatrix} \quad (22)$$

with  $\mathbf{R}\mathbf{d} = \mathbf{d}$ . Note  $\langle \mathbf{c}, \mathbf{d} \rangle = 0$  for  $\mathbf{c}$  and  $\mathbf{d}$  to be unique.

# Screw Motion and Exponential Map

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## Screw Motion and Exponential Map (Brockett, 1990)

Range and null space of a  $\tilde{\omega}$  are orthogonal. Thus,

$$\begin{pmatrix} \mathbf{I} & \mathbf{c} \\ 0 & 1 \end{pmatrix} \begin{pmatrix} \tilde{\omega} & \mathbf{d} \\ 0 & 0 \end{pmatrix} \begin{pmatrix} \mathbf{I} & -\mathbf{c} \\ 0 & 1 \end{pmatrix} = \begin{pmatrix} \tilde{\omega} & \mathbf{d} - \tilde{\omega}\mathbf{c} \\ 0 & 0 \end{pmatrix} \quad (23)$$

establishes that every motion of the form  $\begin{pmatrix} \tilde{\omega} & \mathbf{d} \\ 0 & 0 \end{pmatrix} \theta$  is a **screw motion w.r.t some origin.**

# Group Composition and Screws Connection

## Screw Theory

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## The Lie Algebra Representation, $\mathfrak{se}(3)$

Observe:

$$\tilde{(\cdot)}_{SO(3)} : \mathbb{R}^3 \rightarrow \mathfrak{so}(3) \quad (24)$$

$$\tilde{(\cdot)}_{SE(3)} : \mathbb{R}^6 \rightarrow \mathfrak{se}(3) \quad (25)$$

$\tilde{\omega}(S) \in \mathfrak{se}(3)$ : **Twist parameterization** of a curve, deformation, **screw**.

$\omega(S) \in \mathbb{R}^6$ : **Motion vector** e.g. linear + angular velocities, axial, shear, bending, and torsion motion.